

COMPUTER READABLE STORAGE MEDIUM WITH INSTRUCTIONS
FOR MONITORING CATALYTIC DEVICE

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Background of the Invention

NOx emission control devices, for example catalysts containing a precious metal and elements such as barium, cesium, and lanthanum, can be used to reduce NOx emissions for lean burn engines, such as direct injection spark ignition engine. These devices store NOx produced during the engine lean operation. Since these devices have a finite NOx storage capacity, it has to be regenerated every once in a while in order to sustain long periods of lean operation. This regeneration can be accomplished with rich engine operation during which the device releases and converts the stored NOx.

These emission control devices can be exposed to environments with high temperature from the engine exhaust gas and also from the exotherm introduced by chemical reactions, such as reduction of stored oxidants. The high temperature exposure over time can reduce oxidant (e.g., O₂ and NOx) storage capacity. In addition to temperature effects, sulfur in the engine exhaust gas can form sulfates on the storage sites, thereby reducing the storage capacity. Both the exhaust emissions and the fuel consumption increase as the storage capacity decreases.

One approach to monitoring the effectiveness of an exhaust emission control device used during lean operation to reduce NOx emissions monitors the rich purging time of rich cycles. Such a device is described in JP 08-232644.

5 The inventors herein have recognized a disadvantage with approaches that simply consider the rich purging time, or only consider the lean NOx storage time of an emission control device. Specifically, simply monitoring the rich purging time results in large deviation of measurement results depending how
10 much NOx was stored in the device, thereby requiring complex algorithms and significant model calibration efforts. Similarly, only monitoring the lean storage time does not take into account variations in NOx releasing/reduction reactions that can change as the catalyst ages.

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Summary of an Aspect of the Invention

The above disadvantages are overcome by a computer readable storage medium having stored data representing instructions executable by a computer to monitor a catalytic device coupled
20 in an exhaust of an internal combustion engine of a vehicle.

The storage medium comprises:

instructions for determining a lean operation time during which the engine is operated lean;

instructions for determining a rich operation time during which the engine is operated rich;

instructions for determining a ratio between said lean time and said rich time; and

5 instructions for determining degradation of the catalyst based on said ratio.

By using a ratio of the lean and the rich durations, information from both the oxidant storage ability (NOx storage capacity) and the NOx releasing/reduction capability is utilized
10 to determine the functionality of the catalyst. In this way, improved diagnostics can be achieved with reduced calibration complexity and computational load.

Brief Description of the Drawings

15 The advantages described herein will be more fully understood by reading an example of an embodiment in which the invention is used to advantage, referred to herein as the Description of An Embodiment, with reference to the drawings, wherein:

20 Figure 1 is a schematic diagram of an engine wherein the invention is used to advantage;

Figures 2-3 are high level flow charts of routines for controlling the engine and monitoring the emission control system;

Figures 4A-4D are graphs illustrating correction factors for monitoring the emission control system according to an example embodiment of the invention; and

Figures 5-6 show experimental and predicted data illustrating operation of an example embodiment of the invention.

Description of an Embodiment

Direct injection spark ignited internal combustion engine 10, comprising a plurality of combustion chambers, is controlled by electronic engine controller 12 as shown in Figure 1. Combustion chamber 30 of engine 10 includes combustion chamber walls 32 with piston 36 positioned therein and connected to crankshaft 40. In this particular example, piston 30 includes a recess or bowl (not shown) to help in forming stratified charges of air and fuel. Combustion chamber 30 is shown communicating with intake manifold 44 and exhaust manifold 48 via respective intake valves 52a and 52b (not shown), and exhaust valves 54a and 54b (not shown). Fuel injector 66 is shown directly coupled to combustion chamber 30 for delivering liquid fuel directly therein in proportion to the pulse width of signal fpw received from controller 12 via conventional electronic driver 68. Fuel is delivered to fuel injector 66 by a conventional high pressure fuel system (not shown) including a fuel tank, fuel pumps, and a

fuel rail.

Intake manifold 44 is shown communicating with throttle body 58 via throttle plate 62. In this particular example, throttle plate 62 is coupled to electric motor 94 so that the position of throttle plate 62 is controlled by controller 12 via electric motor 94. This configuration is commonly referred to as electronic throttle control (ETC) which is also utilized during idle speed control. In an alternative embodiment (not shown), which is well known to those skilled in the art, a bypass air passageway is arranged in parallel with throttle plate 62 to control inducted airflow during idle speed control via a throttle control valve positioned within the air passageway.

Exhaust gas oxygen sensor 76 is shown coupled to exhaust manifold 48 upstream of catalytic converter 70. In this particular example, sensor 76 provides signal UEGO to controller 12 which converts signal UEGO into a relative air-fuel ratio λ . Signal UEGO is used to advantage during feedback air-fuel ratio control in a manner to maintain average air-fuel ratio at a desired air-fuel ratio as described later herein. In an alternative embodiment, sensor 76 can provide signal EGO (not shown) which indicates whether exhaust air-fuel ratio is either lean of stoichiometry or rich of stoichiometry.

Conventional distributorless ignition system 88 provides ignition spark to combustion chamber 30 via spark plug 92 in response to spark advance signal SA from controller 12.

Controller 12 causes combustion chamber 30 to operate in
5 either a homogeneous air-fuel ratio mode or a stratified air-fuel ratio mode by controlling injection timing. In the stratified mode, controller 12 activates fuel injector 66 during the engine compression stroke so that fuel is sprayed directly into the bowl of piston 36. Stratified air-fuel ratio layers
10 are thereby formed. The strata closest to the spark plug contains a stoichiometric mixture or a mixture slightly rich of stoichiometry, and subsequent strata contain progressively leaner mixtures. During the homogeneous mode, controller 12 activates fuel injector 66 during the intake stroke so that a
15 substantially homogeneous air-fuel ratio mixture is formed when ignition power is supplied to spark plug 92 by ignition system 88. Controller 12 controls the amount of fuel delivered by fuel injector 66 so that the homogeneous air-fuel ratio mixture in chamber 30 can be selected to be substantially at (or near)
20 stoichiometry, a value rich of stoichiometry, or a value lean of stoichiometry. Operation substantially at (or near) stoichiometry refers to conventional closed loop oscillatory control about stoichiometry. The stratified air-fuel ratio mixture will always be at a value lean of stoichiometry, the

exact air-fuel ratio being a function of the amount of fuel delivered to combustion chamber 30. An additional split mode of operation wherein additional fuel is injected during the exhaust stroke while operating in the stratified mode is available. An
5 additional split mode of operation wherein additional fuel is injected during the intake stroke while operating in the stratified mode is also available, where a combined homogeneous and split mode is available.

Catalytic converter 72 can be a Nitrogen oxide (NOx)
10 absorbent or trap, and can be a single brick, or multiple bricks in a single canister, or multiple bricks in multiple canisters. Catalyst 72 is shown positioned downstream of catalytic converter 70. Catalyst 72 absorbs NOx when engine 10 is operating lean of stoichiometry. The absorbed NOx is
15 subsequently reacted with HC and catalyzed during a NOx purge cycle when controller 12 causes engine 10 to operate in either a rich mode or a near stoichiometric mode.

Controller 12 is shown in Figure 1 as a conventional microcomputer including: microprocessor unit 102, input/output
20 ports 104, an electronic storage medium for executable programs and calibration values, shown as read-only memory chip 106 in this particular example, random access memory 108, keep alive memory 110, and a conventional data bus.

Controller 12 is shown receiving various signals from sensors coupled to engine 10, in addition to those signals previously discussed, including: measurement of inducted mass air flow (MAF) from mass air flow sensor 100 coupled to throttle body 58; engine coolant temperature (ECT) from temperature sensor 112 coupled to cooling sleeve 114; a profile ignition pickup signal (PIP) from Hall effect sensor 118 coupled to crankshaft 40 giving an indication of engine speed (RPM); throttle position TP from throttle position sensor 120; and absolute Manifold Pressure Signal MAP from sensor 122. Engine speed signal RPM is generated by controller 12 from signal PIP in a conventional manner and manifold pressure signal MAP provides an indication of engine load.

In this particular example, temperature Tcat of catalytic converter 70 and temperature Ttrp of catalyst 72 are inferred from engine operation as disclosed in U.S. Patent No. 5,414,994, the specification of which is incorporated herein by reference. In an alternate embodiment, temperature Tcat is provided by temperature sensor 124 and temperature Ttrp is provided by temperature sensor 126.

Fuel system 130 is coupled to intake manifold 44 via tube 132. Fuel vapors (not shown) generated in fuel system 130 pass through tube 132 and are controlled via purge valve 134. Purge valve 134 receives control signal PRG from controller 12.

Exhaust sensor 140 is a sensor that produces two output signals. First output signal (SIGNAL1) and second output signal (SIGNAL2) are both received by controller 12. Exhaust sensor 140 can be a sensor known to those skilled in the art that is
5 capable of indicating both exhaust air-fuel ratio and nitrogen oxide concentration.

In one embodiment, SIGNAL1 indicates exhaust air-fuel ratio and SIGNAL2 indicates nitrogen oxide concentration. In this embodiment, sensor 140 has a first chamber (not shown) in which
10 exhaust gas first enters where a measurement of oxygen partial pressure is generated from a first pumping current. Also, in the first chamber, oxygen partial pressure of the exhaust gas is controlled to a predetermined level. Exhaust air-fuel ratio can then be indicated based on this first pumping current. Next,
15 the exhaust gas enters a second chamber (not shown) where NO_x is decomposed and measured by a second pumping current using the predetermined level. Nitrogen oxide concentration can then be indicated based on this second pumping current.

In an alternative embodiment, a port fuel injection engine
20 can be used as engine 10, where fuel is injected through port injectors into intake manifold 44. The port injected engine can operate homogeneously substantially at stoichiometry, rich of stoichiometry, or lean of stoichiometry.

Those skilled in the art will recognize, in view of this disclosure, that the methods of the present invention can be used to advantage with either port fuel injected or directly injected engines.

Referring now to Figure 2, a routine is described for controlling and determining lean and rich operating times. First, in step 210, the routine determines whether lean burn engine operation is enabled. This determination can be based on various engine operating factors, such as for example: time since engine start, exhaust gas temperature, engine coolant temperature, and various other factors. When the answer to step 210 is "yes", the routine continues to step 212. In step 212, the routine operates the engine at a lean air-fuel ratio (generally leaner than approximately 18:1). The lean air-fuel ratio set point is generally determined based on engine speed versus engine load maps. The lean combustion air-fuel ratio value is controlled to set point via feedback from exhaust gas oxygen sensors using a proportional-integral feedback controller. Further, in an alternate embodiment, the desired lean air-fuel ratio can be determined based on a requested engine torque versus engine speed.

Next, in step 214, the routine determines whether to end lean operation based on an operating condition. There are various methods available for making this determination. In one

example, the routine estimates the amount of NOX stored in the emission control device using an estimate of the amount of NOX generated in the engine and cumulatively adding this amount to integrate the generated NOX amount and thereby determine the amount of NOX stored. When this amount of NOX that is estimated to be stored in the emission control device reaches a preset value, the routine indicates that lean operation should be ended.

In an alternate embodiment, the routine utilizes a downstream NOX sensor that detects the NOX concentration exiting the emission control device. The routine determines an amount of exiting NOX per distance traveled by the vehicle, and compares this value to a set point level of emissions per distance. When the measured amount of NOX per distance traveled by the vehicle reaches the threshold, the routine determines that lean operation should be ended. Further, there are various other methods that can be used to determine when to end the lean operation.

In step 216, the routine checks to see whether lean operation should be ended as determined by step 214. When the answer to step 216 is "no", the routine returns to continue monitoring whether to end lean operation in step 214. When the answer to step 216 is "yes", the routine continues to step 218.

In step 218, the routine ends the lean operation and determines the lean operating time of the just ended lean operation. Next, in step 220, the routine commences rich engine operation to purge stored oxidants in the emission control device. The degree of richness at which the engine operates during this rich operating period is determined based on various operating conditions, such as for example: exhaust gas temperature, emission control device temperature, and engine conditions such as engine speed.

Continuing with Figure 2, in step 222, the routine calculates a lean time correction based on the average engine operating conditions during the lean operation. These corrections are described in more detail below with regard to Figures 4B, 4C, and 4D and equations 1-4. Note that only a single correction parameter can be used, or any combination of correction parameters can be used, or all of the lean correction parameters can be used. Then, in step 224, the routine determines conditions downstream of the emission control device to monitor for reductant (HC and CO) breakthrough. Reductant breakthrough downstream of the emission control device during rich operation indicates that a significant portion of the stored oxidants (oxygen and NOX) have been released and reduced across the catalytic surface of the emission control device. There are various methods for monitoring whether reductant

breakthrough has occurred, such as, for example: using an exhaust gas oxygen sensor downstream of the emission control device and monitoring whether the sensor indicates a transition from a lean to a stoichiometric or a rich condition, or whether the sensor indicates a transition from a stoichiometric to a rich operating condition.

Then, in step 226, the routine determines whether rich operation should be ended based on the detected breakthrough in step 224. When the answer to step 226 is "yes", the routine continues to step 228. When the answer to step 226 is "no", the routine returns to step 224 to continue monitoring the conditions downstream of the emission control device.

Continuing with Figure 2, in step 228 the routine ends the rich operation and determines the rich operating time of the just ended rich operation. Then, in step 230, the routine calculates rich time corrections based on the average conditions during the rich operation and utilizing the graphs described in Figures 4A, 4C, and 4D and equations 1-4 (described below). Finally, in step 232, the routine calculates the ratio of the corrected lean and corrected rich times and stores this ratio as a function of the corrected lean time in the computer memory. Note that the routine could, in an alternate embodiment, simply use each calculated ratio at the end of a lean rich cycle to calculate an average ratio for the emission control device.

Alternatively, the ratio can be stored across different operating conditions such as, for example, the rich operating time.

Referring now to Figure 3, a routine is described for
5 determining degradation of the emission control device based on the calculated ratio. First, in step 310, the routine determines whether degradation detection of the emission control device is enabled based on current operating conditions. For example, degradation detection may not be enabled during initial
10 engine starting, or various high speed high load engine operating conditions. Further, the degradation detection is generally enabled only during lean burn operation as the routine utilizes the lean/rich operating time ratio as described above herein with regard to Figure 2. Note however, that degradation
15 detection can be enabled during other operating conditions if desired.

When the answer to step 310 is "yes", the routine continues to step 312. In step 312, the routine determines whether the ratio determined in step 232 has been determined for a
20 sufficient number of lean/rich cycles, and for a sufficient range of corrected lean times. When the answer to step 312 is "yes", the routine continues to step 314.

In step 314, the routine calculates an average error (e) between the expected and determined ratios across the corrected

lean times. In this way, the diagnostic routine utilizes information for a variety of lean operating times to determine degradation of the emission control device. Further, such an approach allows the routine to utilize expected emission control
5 device performance that is efficiently stored as an expected ratio for various lean operating times.

Next, in step 316, the routine determines whether the average error is greater than a threshold value (LIMIT) and whether the flag (FLAG) is set to "1". Note that upon
10 initialization, the flag (FLAG) is preset to "1". When the answer to step 316 is "yes", the routine continues to step 318 to perform a sulfur decontamination by operating the engine at increased exhaust gas temperatures and oscillating the inlet air-fuel ratio to the emission control device about
15 stoichiometry to thereby remove sulfur. Next, in step 320, the routine sets the FLAG to "zero". In this way, the routine first attempts to utilize sulfur decontamination to rejuvenate the functionality of the emission control device.

When the answer to step 316 is "no", the routine continues
20 to step 322 to determine whether the air is above the LIMIT value and the FLAG is set to "zero". When the answer to step 322 is "yes", the routine continues to step 324 to indicate degradation of catalyst 72 to the driver. When the answer to

either steps 322, 312, or 310 is "no", the routine ends. In summary, the routine implements a diagnostic method that:

- calculates R_T after each lean-rich cycle (after the rich operation is over) using equations (1)-(3) (described below).
- calculate \bar{R}_T using the model of equation (4) (described below).
- Calculate difference $e = R_T - \bar{R}_T$
- If e becomes greater than a predefined threshold, activate deSOx process (which is to remove sulfate formed in catalyst 72) to recover NOx storage capacity.
- If e becomes greater than a predefined threshold even after deSOx, indicate degradation of catalyst 72.

In this way, the engine controller is able to utilize information from both the lean operating (NOx storage) conditions and the rich operating conditions (NOx release and reduction) in order to more accurately determine the functionality of the emission control device. Thus, the catalyst monitoring algorithm monitors the ratio between the duration of lean engine operation and the duration of rich engine operation. The ratio is then used to characterize the "health" status (normal catalyst) as a degraded catalyst has a

different characteristic from that of the normal catalyst.

Then, when this difference become significant and exceeds a threshold, a degraded device is declared and indicated to the vehicle driver. In this way, on-board emission measurements are reduced and a simplified approach is obtained. This reduces strategy complexity and on-board computational requirements.

Model/Algorithm Details

The Normalized Storage Purge Ratio (NSPR): R_T is defined by equations 1-3, below:

$$R_T = \frac{T_L^*}{T_R^*} \quad (1)$$

$$T_L^* = T_L \times \theta_{NOx_L} \times \theta_{MAF_L} \times \theta_{T_LNT_L} \quad (2)$$

$$T_R^* = T_R \times \theta_{AFR_R} \times \theta_{MAF_R} \times \theta_{T_LNT_R} \quad (3)$$

where T_L^* : normalized lean duration [seconds]

T_R^* : normalized rich duration [seconds]

T_L : lean duration [seconds]

T_R : rich duration [seconds]

θ_{AFR_R} : normalization factor for air-fuel ratio during the rich operation (Figure 4A)

θ_{NOx_L} : normalization factor for NOx concentration in-
coming to LNT during the lean operation (Figure 4B)

$\theta_{MAF_L,R}$: normalization factor for exhaust mass flow rate
during the lean/rich operation (Figure 4C)

5 $\theta_{T_LNT_L,R}$: normalization factor for LNT temperature during
the lean/rich operation (Figure 4D)

For a baseline, non-degraded catalyst, R_T is found over
engine and catalyst operation modes with a normal (healthy)
10 catalyst. A model is found based on experimental data and
testing for R_T as a function of the lean engine duration and
operating conditions using equation 4. This model and data is
then stored in engine controller memory

$$\bar{R}_T = f(T_L^*, \theta_{AFR_R}, \theta_{MAF_R}, \theta_{T_LNT_R}) \quad (4)$$

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Referring now to Figures 4A through 4D, various correction
factors are illustrated for either the lean operating time or
the rich operating time, or both. The factors are stored in
maps as a function of conditions as indicated in the Figures.
20 Specifically, Figure 4A illustrates a rich correction factor as
a function of the rich air-fuel ratio during the rich operating.
Figure 4B illustrates a lean time correction factor as a
function of the incoming NOX concentration to the emission

control device. Figure 4C illustrates both a lean and a rich correction factor as a function of the mass airflow sensor (e.g., exhaust flow rate). Finally, Figure 4D illustrates both a lean and a rich correction factor as a function of exhaust gas temperature, or emission control device temperature.

Experimental results of the approach described in Figures 2-4 are shown in Figure 5-6. Specifically, Figure 5 shows modeled and measured ratios for a full capacity catalyst and a $\frac{1}{4}$ capacity catalyst. Clear separation between the data can be seen, and is especially evident via the graph of Figure 6. Increased sensitivity to degradation is obtained for increased lean operating times. As such, in one example embodiment of the present invention, the error (e) is only calculated and utilized for cycles where the corrected lean time is greater than a threshold value, e.g., 50 seconds.

Although several examples of embodiments which practice the invention have been described herein, there are numerous other examples which could also be described.